

# 2010 ANSYS SOUTH AMERICAN CONFERENCE & ESSS USERS MEETING

October 19-22 - Atibaia, SP - Brazil

## Centrifugal Pumps Mechanical Design ANSYS analysis Vibration in Vertical Pumps

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# PRESENTATION PARTS

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- Sulzer , Sulzer Pumps and Sulzer Pumps in Brazil
- Pumps and Vertical Pumps
- Case Study: Vertical Pump in VCP – 3 Lagoas MS
- Conclusion

# Sulzer and Sulzer Pumps



**Sulzer Pumps**



Pumping solutions  
and services



**Sulzer Metco**



Surface technology  
solutions and services



**Sulzer Chemtech**



Components and services  
for separation columns and  
static mixing



**Sulzer Turbo Services**



Service and repair of  
thermal turbomachinery







**Sulzer Innotec**



Contract research and  
technical services

# Sulzer market

	Sulzer Pumps	Sulzer Metco	Sulzer Chemtech	Sulzer Turbo Services
 <p>Oil and gas (incl. HPI, CPI, etc.)</p>	■	■	■	■
 <p>Pulp and paper</p>	■	■	■	■
 <p>Turbines and power generation</p>	■	■	■	■
 <p>Automotive</p>		■	■	

# Sulzer Brasil - Jundiaí



**Pattern Storage**  
• 1400m<sup>2</sup>

**Office building**  
• 2500m<sup>2</sup>  
• 2 stories

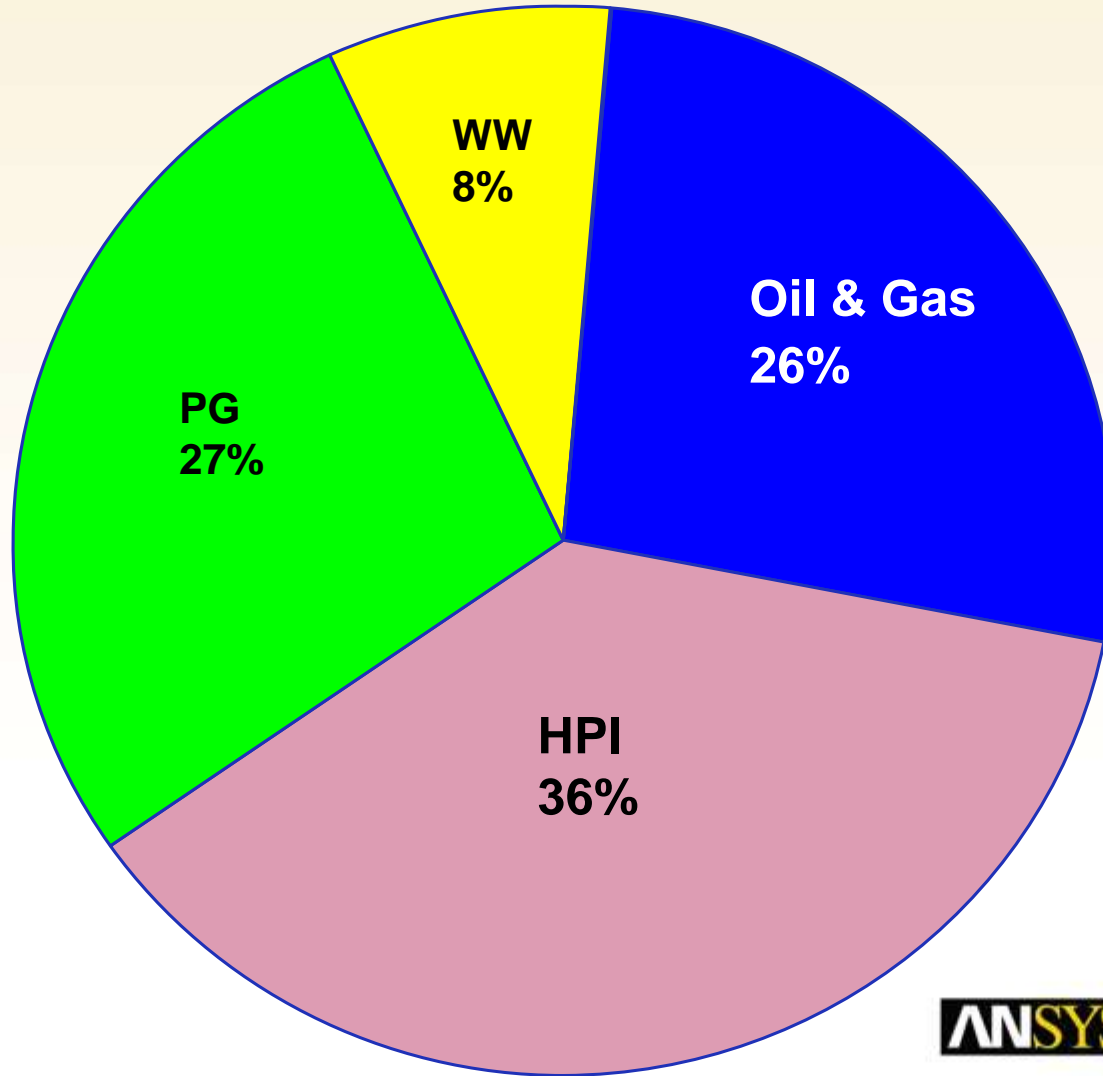
**Factory**  
• 7200m<sup>2</sup>  
• 5 bays  
• 45t lifting cap.

**Canteen**

**Component Machining**  
• 2000 m<sup>2</sup>

**Foundry**  
• 3500 m<sup>2</sup>

# Sulzer Brasil - 2009



Orders received  
New Pumps  
360 BRL million

# Sulzer Pumps Products – except Vertical

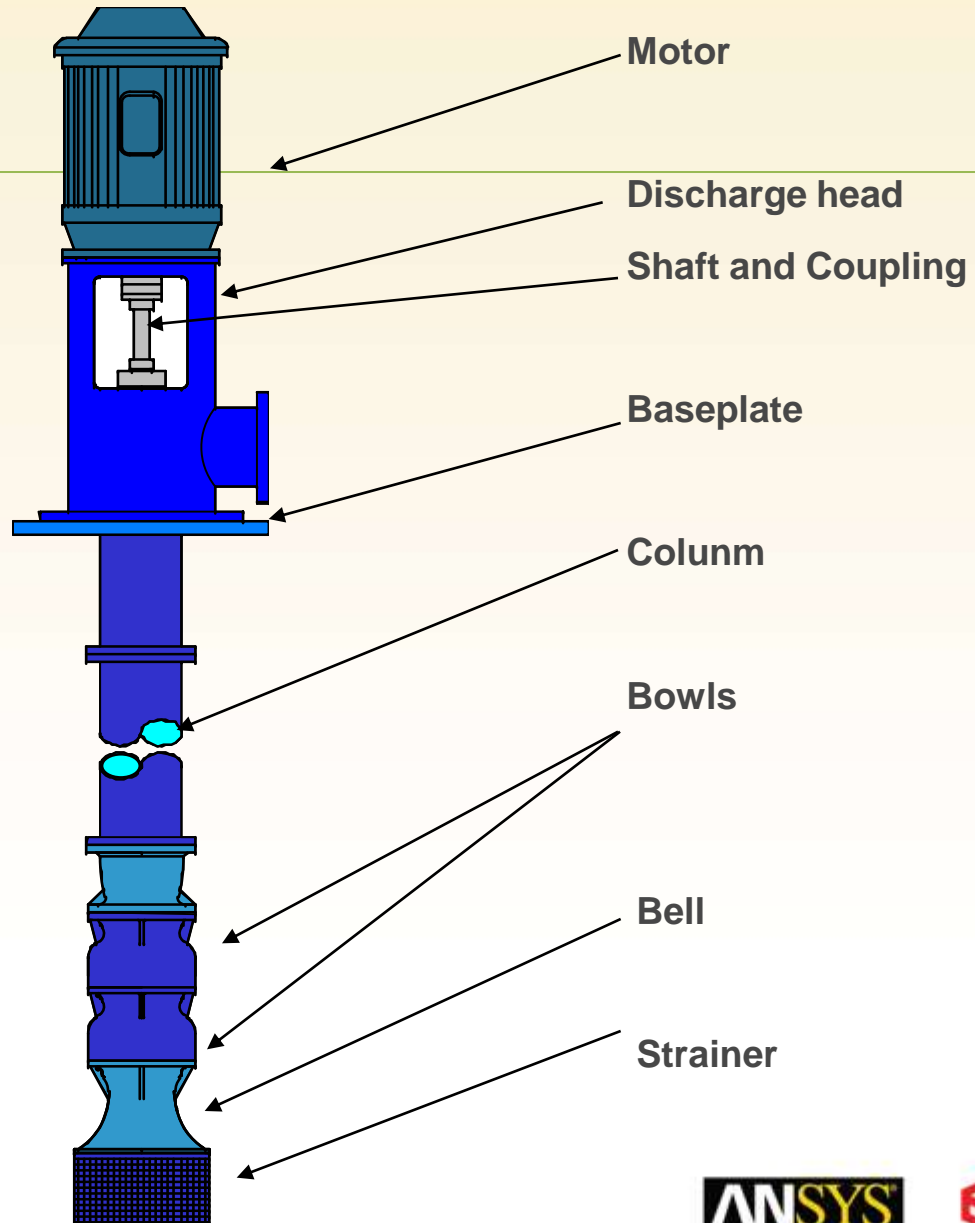
<b>Overhung</b>	<b>Horizontal</b>	<i>Foot Mounted</i>	<b>CPT, Z</b>
		<i>Centerline Mounted</i>	<b>CAP, OHH, OHL</b>
	<b>Vertical</b>	<i>In-Line</i>	<b>OHV</b>
		<i>End-Suction</i>	<b>ZAV</b>
<b>Between Bearings</b>	<b>1 &amp; 2 stages</b>	<i>Axially Split</i>	<b>SMN, SMH, HSB, HPDM, ZPP</b>
		<i>Radially Split</i>	<b>BBT, BBS, BBT-D, CD, BBS-SC</b>
	<b>Multistage</b>	<i>Axially Split</i>	<b>MSD, MSE, MSD2</b>
		<i>Radially Split-Single casing</i>	<b>MBN, MC, MD, ME</b>
		<i>Radially Split-Double Casing - Barrel</i>	<b>GSG, CP, HPcp, HPT</b>

# Sulzer Vertical Pumps

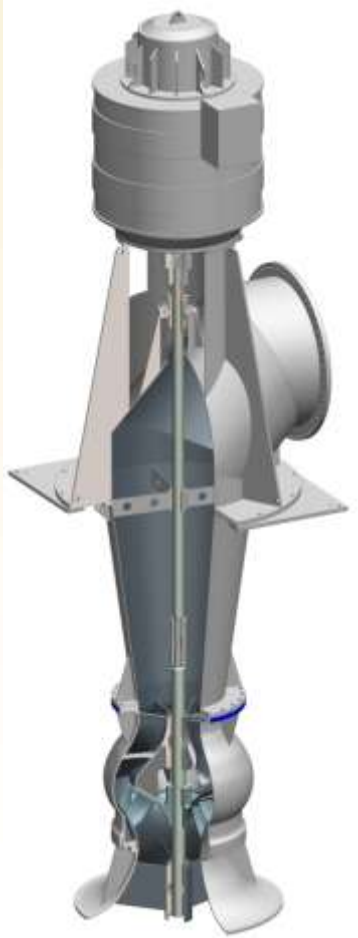
<b>Single Casing</b>	<b>Through Column</b>	<i>Diffuser</i>	<b>TMC BK, BK<sub>n</sub>, BSm SJT, SJM, JTS</b>
		<i>Volute</i>	<b>BSD</b>
		<i>Axial Flow</i>	<b>BP<sub>n</sub> SJP</b>
	<b>Separate Discharge</b>	<i>Line-shaft</i>	<b>ZN</b>
		<i>Cantilever</i>	<b>NKP</b>
<b>Double Casing</b>		<i>Diffuser</i>	<b>TTMC BKC, SJD VCR</b>
		<i>Volute</i>	<b>BDC</b>



# Sulzer Vertical Pump



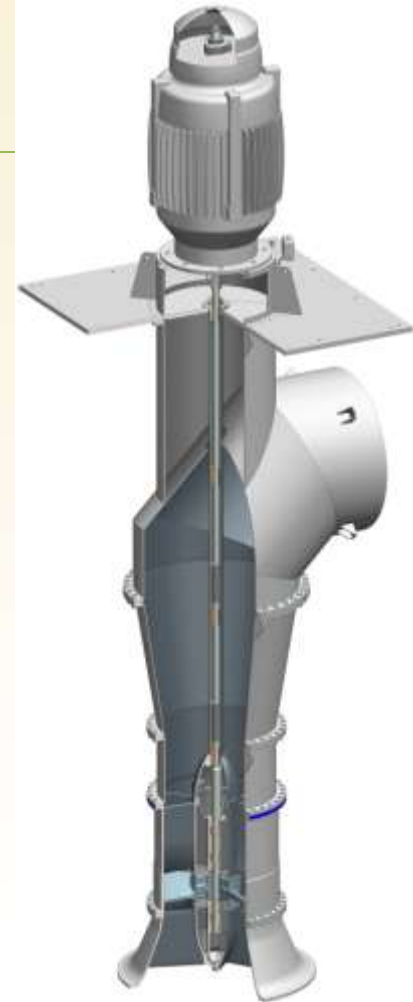
# Sulzer Vertical Pumps – Main Types



**SJT Turbine**  
Ns 1800 < 5000  
nq 35 < 110



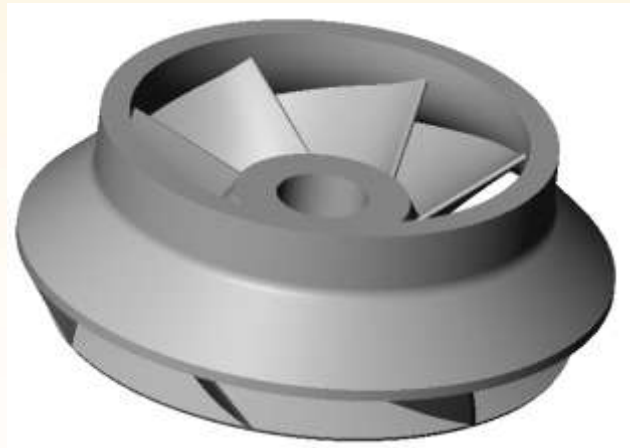
**SJM Mixed Flow**  
Ns 5800 < 8300  
nq 113 < 161



**SJP Axial Flow**  
Ns ~ 14,500  
nq ~ 280

# Vertical Pumps - Hydraulics

## Impeller types



**CLOSED**



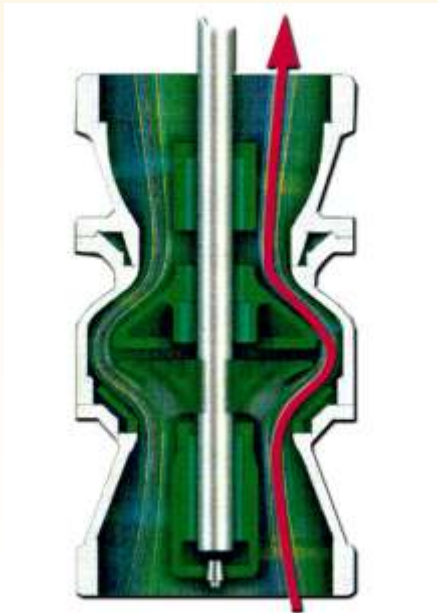
**SEMI-CLOSED**



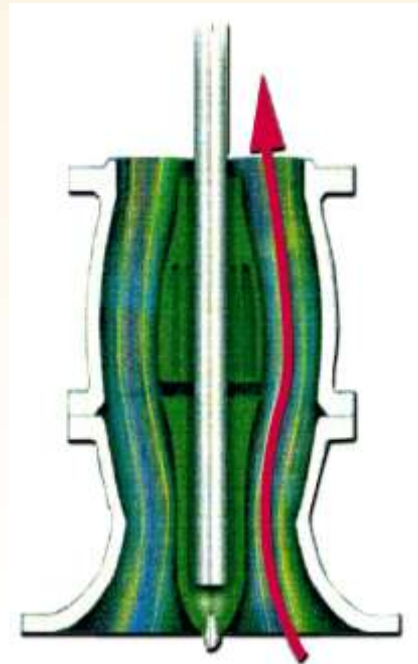
**OPEN**

# Vertical Pumps - Hydraulics

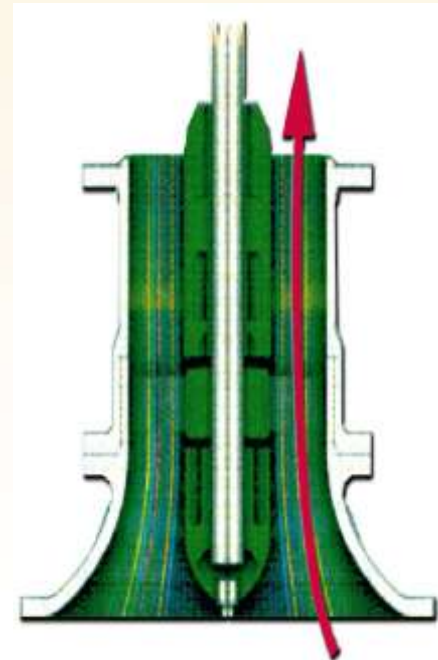
Turbine



Mixed



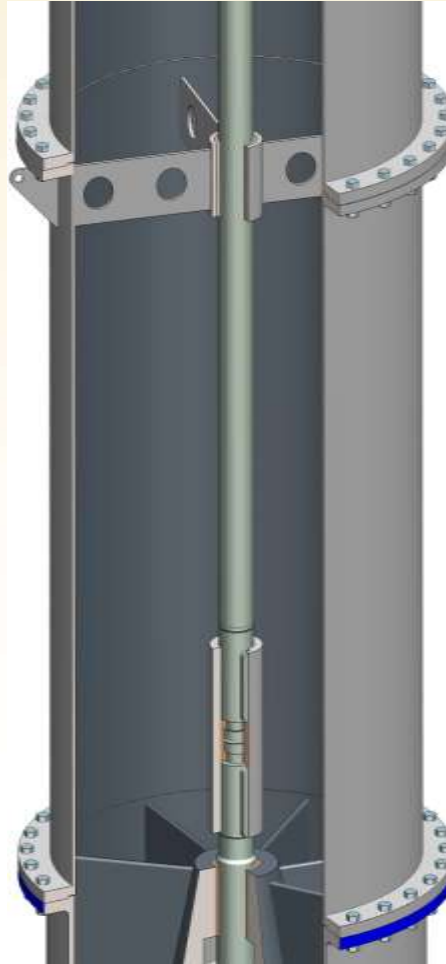
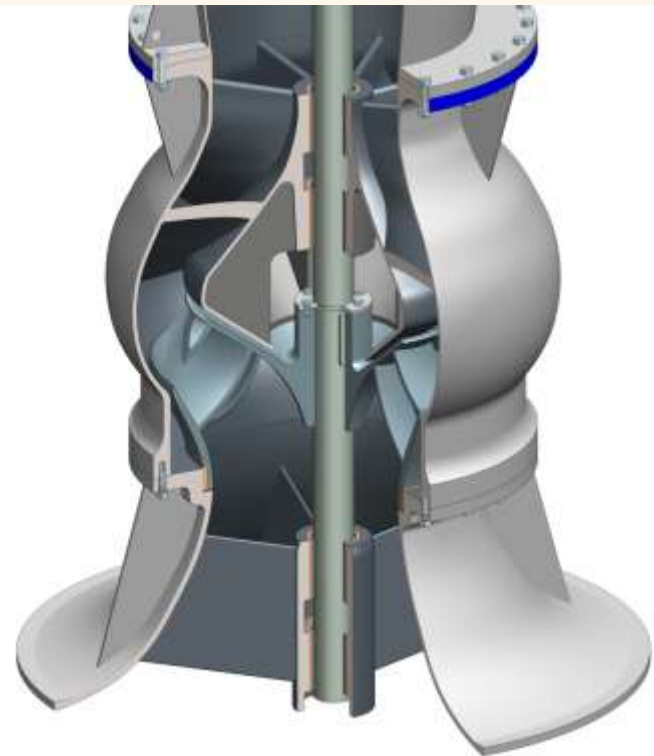
Axial



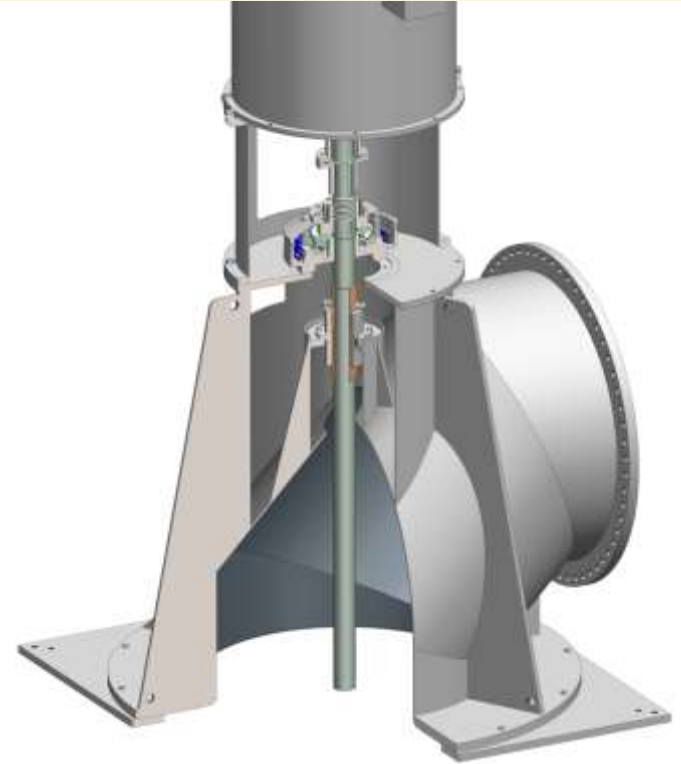
Types according to flow type

# Sulzer Vertical Pumps – 3 main parts

**SUCTION - BOWL**

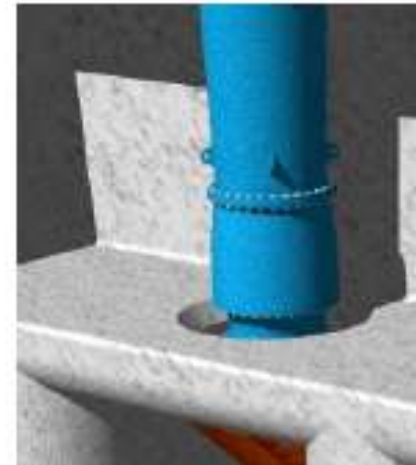
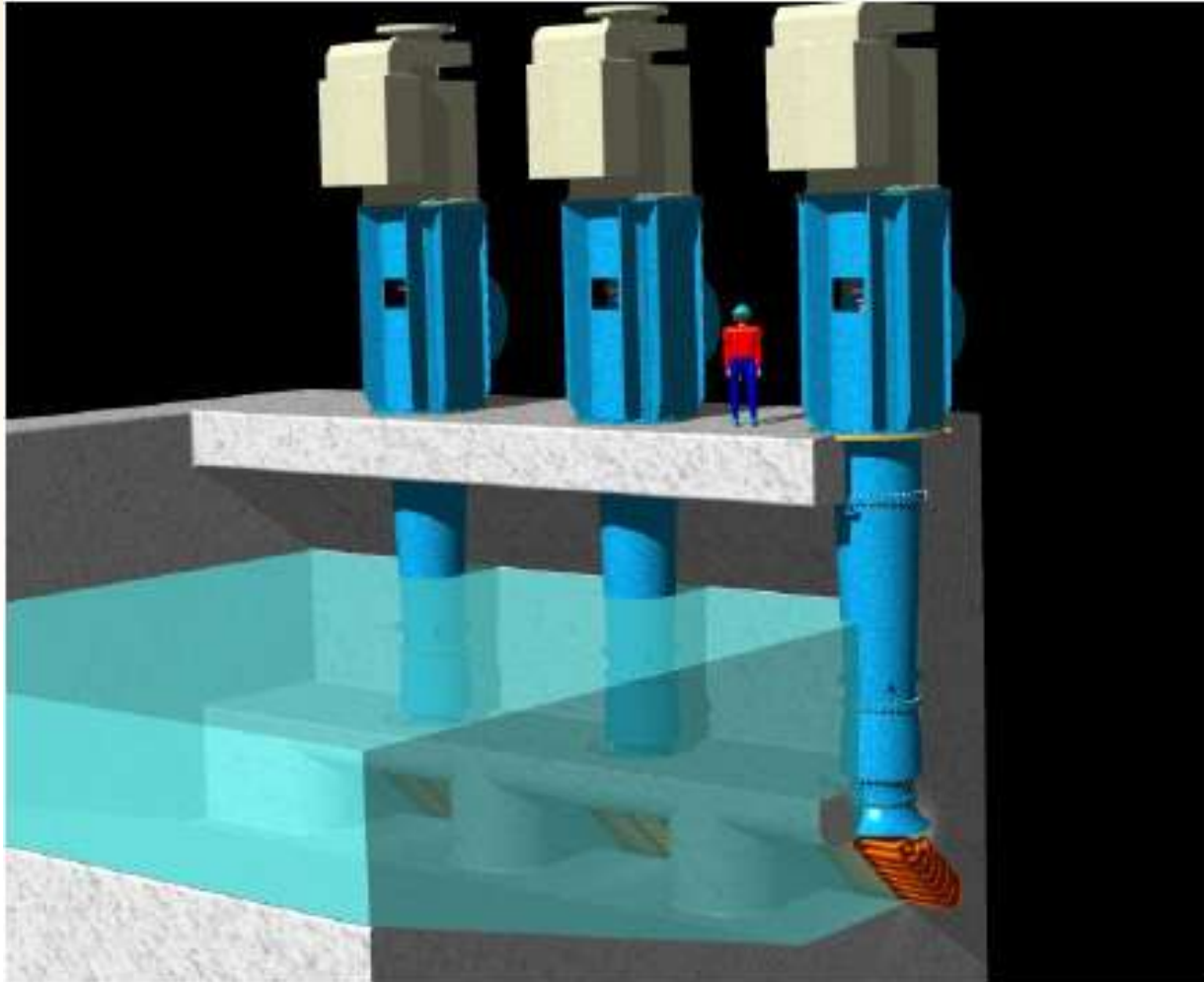


**COLUMN**



**DISCHARGE HEAD**

# Vertical Pumps – Field Installation



# Sulzer Vertical Pump – BSm at test



Cooling System

# Sulzer Vertical Pump – BSm at field



**BSm 1400-1s**

**36'000 m<sup>3</sup>/h at 14m**

**310 RPM, 1520 kW**

**Diam. Discharge**

**2000mm**

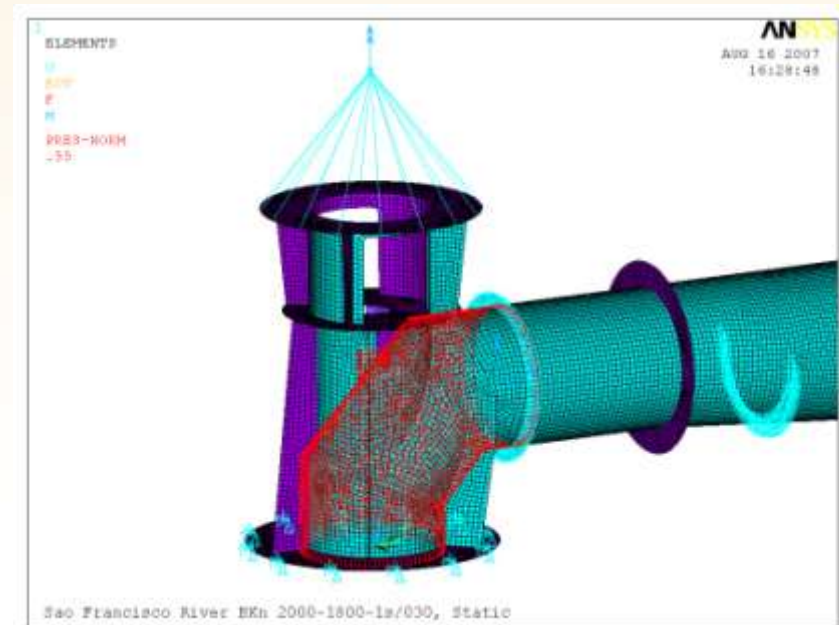


# Vertical Pump – Static Analysis

→ **Example: São Francisco, BK<sub>n</sub>2000-1800-1s/030**

Loads and Restraints:

- **Internal pressure**, applied on "wet surface"
- Pressure on "cover faces" is applied as equivalent axial force
- **Motor torque**
- **Nozzle loads**
- **Rotor mass** at axial bearing
- Restraints at bolt locations and pipe



# Vertical Pump – Static Analysis

→ **Example: São Francisco, BK<sub>n</sub>2000-1800-1s/030**

Allowable Membrane Stress:  $\sigma_{m,all} = \frac{R_{p,0.2}}{SF}$

- Primary Membrane Stress

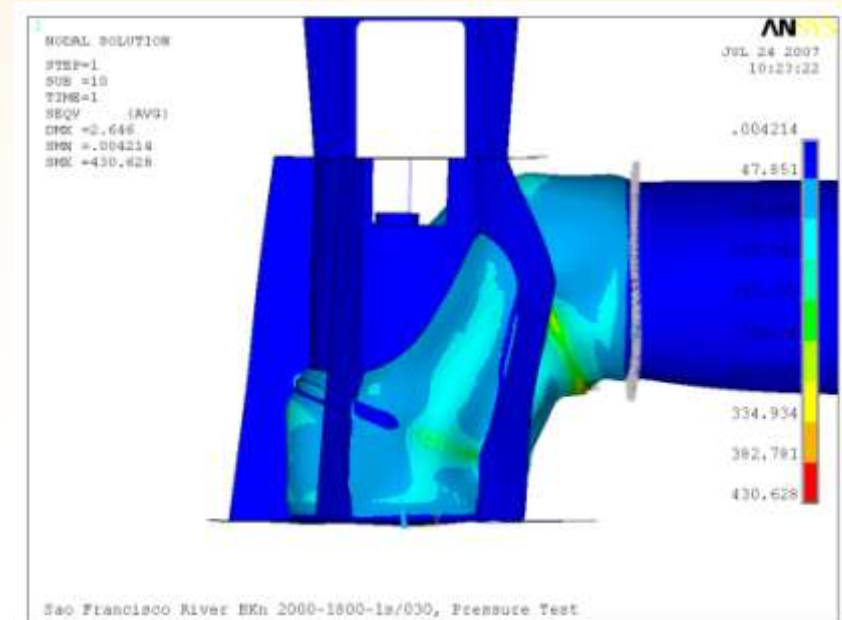
$$\sigma_{eq} \leq \sigma_{m,all}$$

- Local Primary Membrane Stress

$$\sigma_{eq} \leq 1.5 \cdot \sigma_{m,all}$$

- Secondary Membrane + Bending Stress

$$\sigma_{eq} \leq 3 \cdot \sigma_{m,all}$$



# Case Study - Vertical Pump in 3 Lagoas

## General Dimensions

$D_{nom} = 508$  [mm]  
 Height of Column Pipe = 14500 [mm]  
 Height of Motor Stool = 1985 [mm]  
 Height of Motor = 2500 [mm]  
 Total Height = 18985 [mm]

## Performance Data

Rated Flow 2218 [m<sup>3</sup>/h]  
 Rated Head 49 [m]  
 Rated Speed 1186 [rpm]  
 Pumped Fluid Water

## Masses and Inertias

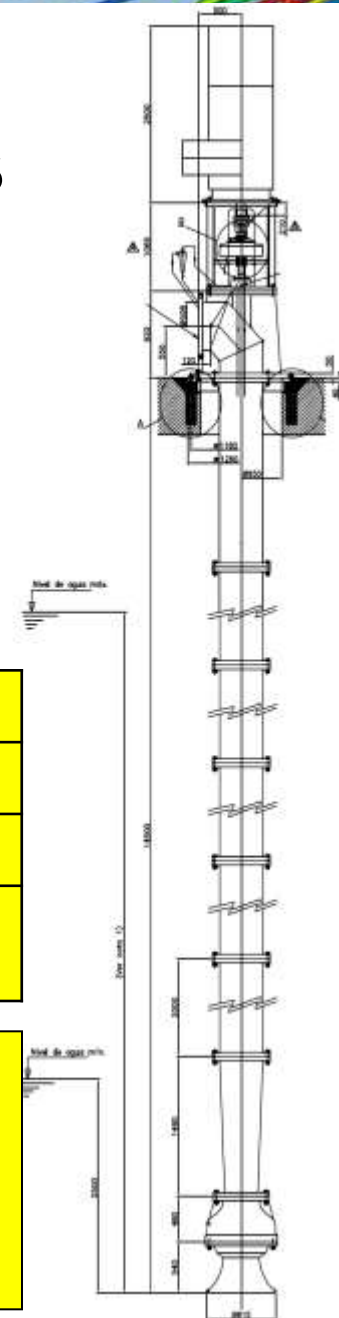
### Material Data

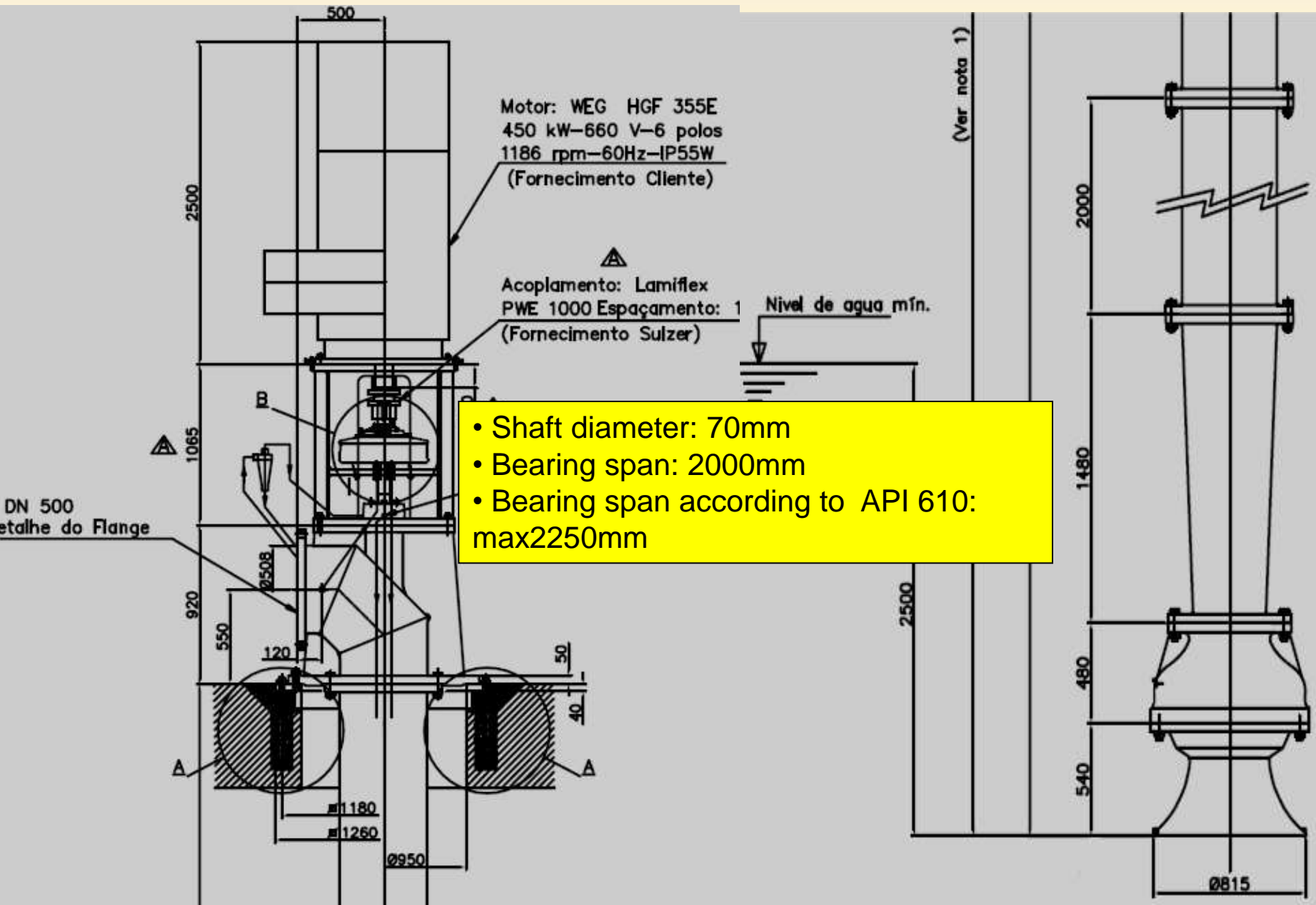
Pipes: A36  
 Flanges: A36  
 Motor Stool: A36  
 Pump Bowl: 0.7040  
 Bellmouth: 0.6025

## Pump Data Geometry

	m [kg]	$I_p$ [kgm <sup>2</sup> ]	$I_t$ [kgm <sup>2</sup> ]
Impeller	165.9	7.23	3.85
Motor	2'790	17.6	-
Axial Bearing	76.9	0.57	-

Other data	Impeller diameter: 567mm - nq 47		
Weights (kg):	Pump: 5400 - Baseplate: 256		
	Motor: 2790 - Total: 8446		



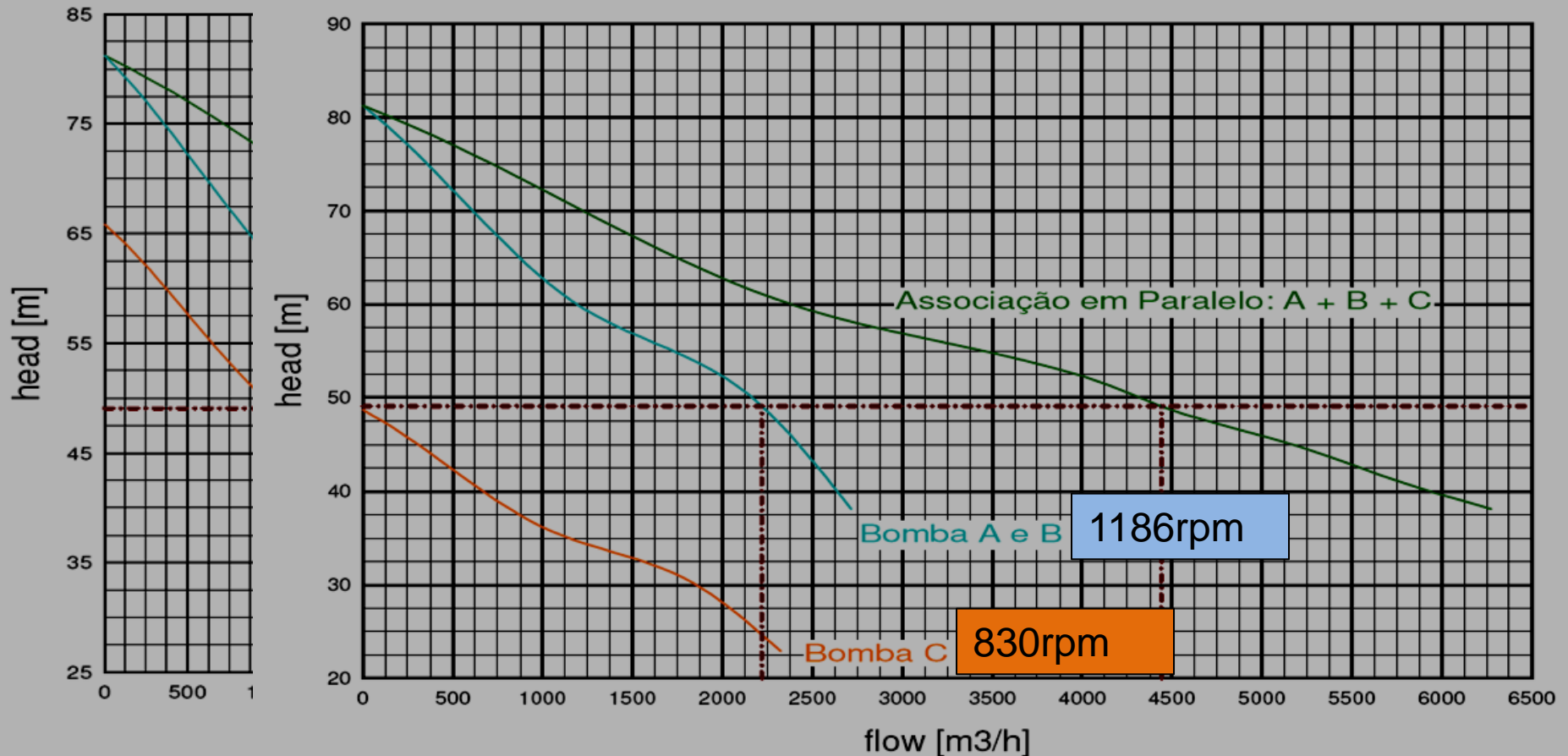


- Shaft diameter: 70mm
- Bearing span: 2000mm
- Bearing span according to API 610: max2250mm

# Case Study - Vertical Pump in 3 Lagoas

Pumps A and B speed 100% = 1186rpm = 19,8Hz => 2218m<sup>3</sup>/h – 49m – 370kW

Pump C speed 70% - 80% - 90% => 13,8Hz - 15,8Hz - 17,8Hz (1067rpm) => 1035m<sup>3</sup>/h



# Case Study - Vertical Pump in 3 Lagoas



B: Modal (min Water, free)  
 Cylindrical Support  
 Frequency: N/A  
 26.02.2010 13:20

A Displacement  
 B Cylindrical Support: 0, m



## Pump Min. Operating Water Level

Rising Pipe Natural Frequencies [Hz]			
	about Y-axis	about X-axis	Figure
1. Mode (1 <sup>st</sup> classic bending mode)	1.1	1.1	Figure B-1
2. Mode (2 <sup>nd</sup> classic bending mode)	6.7	6.8	Figure B-2
3. Mode (3 <sup>rd</sup> classic bending mode)	18.6	18.7	Figure B-3
4. Mode (4 <sup>th</sup> classic bending mode)	35.1	35.2	Figure B-4

**Model: without shaft, sleeves and intermediary rubber bearings.**

Water mass added to model





# Case Study - Vertical Pump in 3 Lagoas



## Pump Min. Operating Water Level

Rising Pipe Natural Frequencies [Hz]			
	about Y-axis	about X-axis	Figure
1. Mode (1 <sup>st</sup> classic bending mode)	0.9	0.9	Figure C-1
2. Mode (2 <sup>nd</sup> classic bending mode)	5.5	5.6	Figure C-2
3. Mode (3 <sup>rd</sup> classic bending mode)	14.2	14.2	Figure C-3
4. Mode (4 <sup>th</sup> classic bending mode)	36.1	36.3	Figure C-4

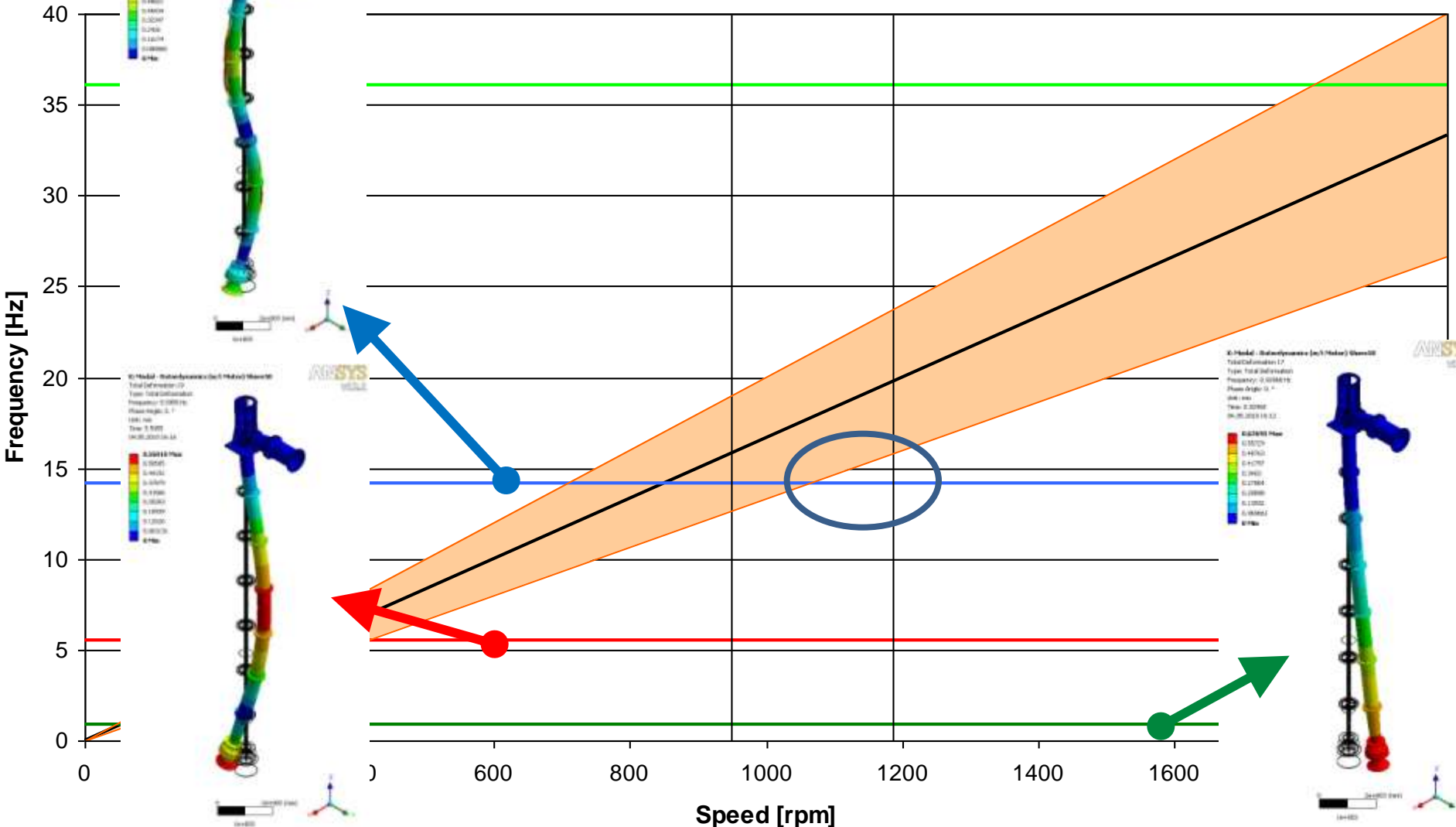
**Model: with shaft, sleeves and intermediary rubber bearings, but not Rotordynamic effect included**

Water mass added to model



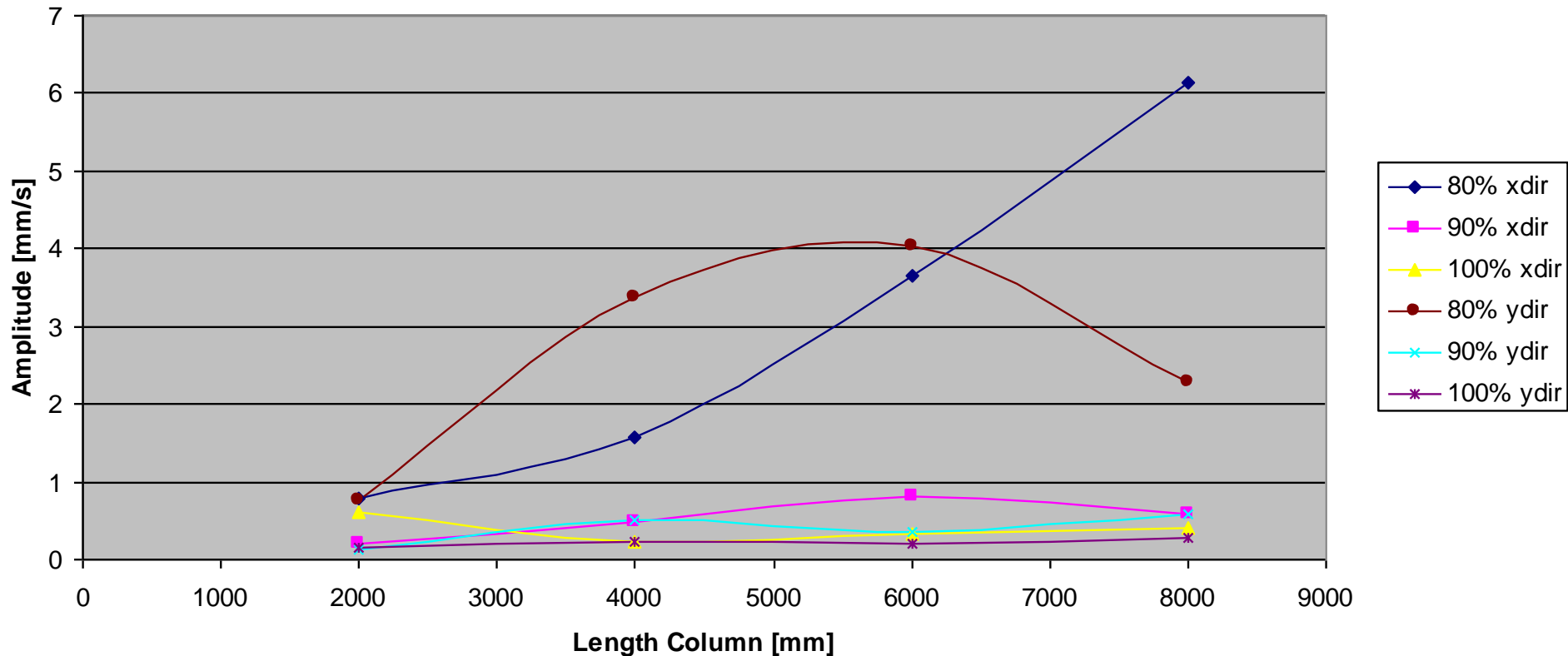
**With shaft, sleeves and rubber bearings**

## Campbell Diagram



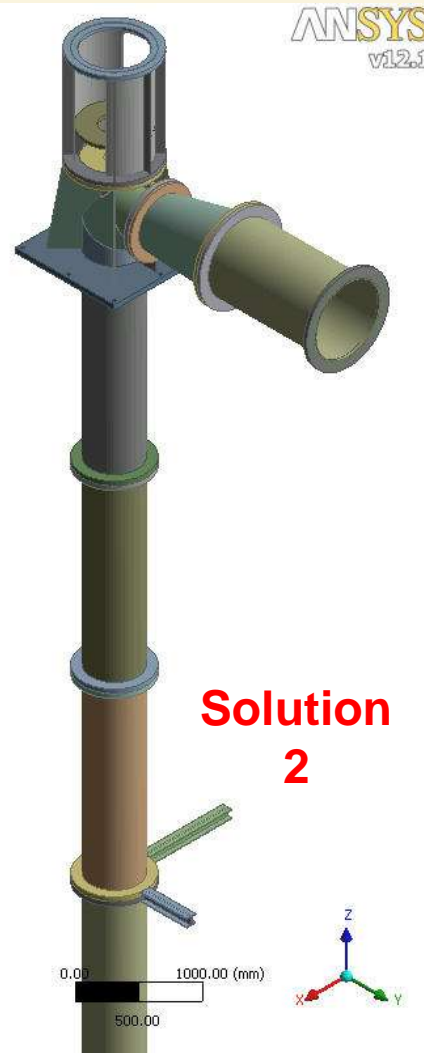
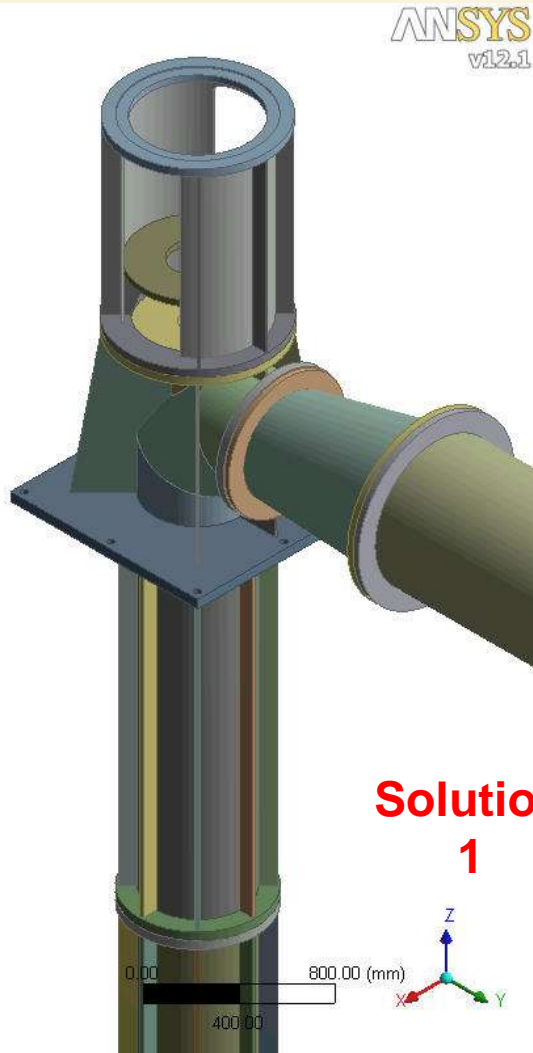
# Case Study - Vertical Pump in 3 Lagoas

Vibration at 5.5Hz



Vibration measurements at field: No resonance at 90% rated speed, but high vibration at 80% rated speed

# Case Study - Vertical Pump in 3 Lagoas

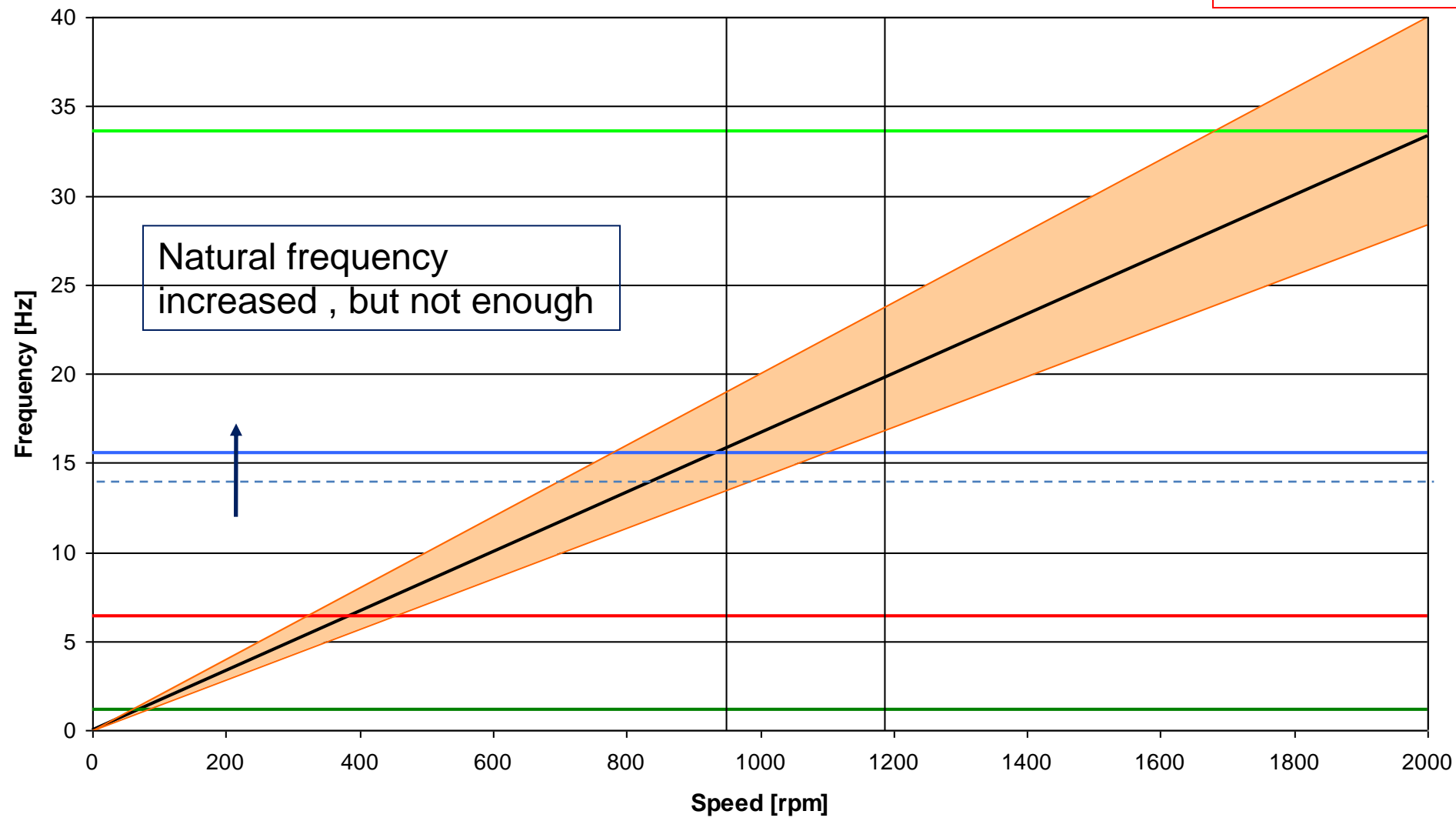


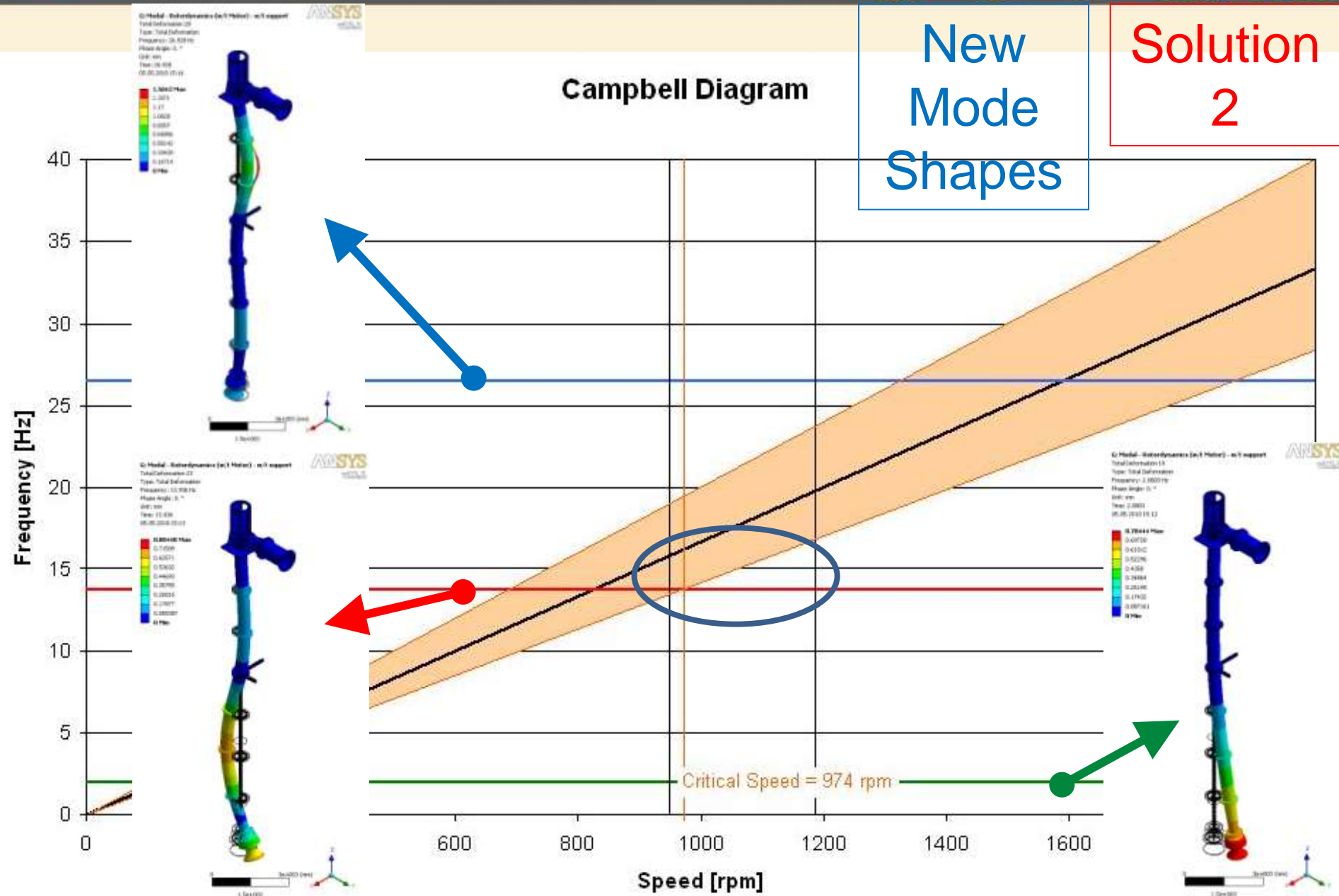
## *Two possible solutions:*

- 1) Increase Natural frequency from ~14Hz to above 23Hz, by adding ribs in the column in order to increase stiffness
- 2) Modify Mode shape, by fixing column to concrete structure

Solution  
1

### Campbell Diagram





# Case Study - Vertical Pump in 3 Lagoas

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## Conclusions

1. Learning: guidelines to Sales => orientation to customer in case of pumps in parallel with different speeds;
2. Optimum model => closest to reality, that means, enough information that explain what is happening at field;
3. To modify natural frequency taking in account separation margin: not ever to increase, but according to each configuration of project.

## Case Study - Vertical Pump in 3 Lagoas

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Gracias

Merci

Obrigado

Danke

Thank you

Grazie